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International Journal of Modern Physics C **World Scientific** Vol. 29, No. 9 (2018) 1850117 (15 pages) w.worldscientific.com © World Scientific Publishing Company 1 DOI: 10.1142/S0129183118501176 23 4 56 Rarefied transitional flow through diverging nano and 7 microchannels: A TRT lattice Boltzmann study 8 9 10 Soroush Fallah Kharmiani and Ehsan Roohi* 11 Department of Mechanical Engineering Ferdowsi University of Mashhad 12Mashhad 91779-1111, Iran 13 *e.roohi@ferdowsi.um.ac.ir 1415Received 4 September 2018 Accepted 30 October 2018 16Published 1718Rarefied isothermal gaseous flow through long diverging micro and nanochannels is investigated 19in this paper using the two-relaxation-time (TRT) lattice Boltzmann method (LBM). The 20simulations are performed over a wide range of Knudsen number, pressure ratio, and divergence angle. The Bounce-Back Specular Reflection (BSR) slip boundary condition is applied and is 21 connected to the second-order slip boundary condition coefficients by means of the antisym-22metric relaxation time and the bounce-back portion parameter. The effects of the slip coeffi-23cients on the wall and centerline Mach numbers, as well as the mass flow rates, are investigated. 24The numerical results are validated with those of the direct simulation Monte Carlo (DSMC) reported in the literature. The results show that the local pressure distributions are almost 25independent of the slip coefficients with excellent agreements with DSMC over a wide range of 26the divergence angle. Our results demonstrate that there is a specific divergence angle at each 27pressure ratio where the local unbounded Knudsen and, as a result, Mach numbers remain constant along the channel. This observation is almost independent of the slip coefficients, 28and the underlying reason is that the pressure drop is compensated by an increase in the 29channel area. 30 Keywords: Divergent microchannel; TRT LBM; rarefied flow; BSR slip boundary condition; 31model validation. 32 33 34 1. Introduction 35The rarefied gaseous flow through nanochannels with sudden or gradual expansion 36 has widespread engineering applications, such as hard disk drives. While consider-37 able work is carried out on investigating the rarefied gas flow through micro and 38 nanochannels with uniform cross-sections, e.g. Refs. 1–7, much fewer researches are 39 reported on microchannels with nonuniform cross-sections. Based on the kinetic 40equation, Sharipov and Bertoldo⁸ proposed a method for determining the mass flow

41 rate through a long tube with a variable radius. Stevanovic⁹ analytically investigated

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1 the 2D microgaseous flow through slowly varying microchannel using the perturbation method. Hemadri et al.¹⁰ experimentally investigated the rarefied gaseous 2flow through diverging microchannel and existence of the Knudsen minimum was 3 reported. Akbari et al.¹¹ proposed a model for predicting the pressure loss in non-4 uniform microchannels with arbitrary cross sections in the slip regime. Ebrahimi and 5Roohi¹² simulated the 2D rarefied gas flow through a diverging microchannel using 6 the DSMC method. Kiselev et al.¹³ studied the supersonic gas flows in radial 7 micronozzles. Darbandi and Roohi¹⁴ investigated the rarefied gaseous flow through 8 9backward-facing steps and reported that the separation length is highly reduced as the rarefaction is increased approaching the transition regime. Mahdavi et al.¹⁵ and 10 Mahdavi and Roohi¹⁶ further studied rarefied flow through step geometries using 11 the DSMC method. Recently, Gavasane et al.¹⁷ also numerically investigated the 1213rarefied gas flows in backward facing microstep using the DSMC method.

As an alternative numerical method, the lattice Boltzmann method $(LBM)^{18-22}$ 1415takes advantage of relative simplicity, computational efficiency, and simulating flows 16involving complex geometries such as the porous media. Furthermore, the kineticbased nature of the LBM makes it a robust tool for simulating complex physics such 17as multi-phase, $^{23-25}$ and micro/nano gaseous flows 26,27 including the shale gas flow. 18 The method is successfully applied to the simulation of the rarefied gas flows between 1920two flat walls up to the end of the transition regime. Wang et $al.^{28}$ reviewed the application of the LBM for simulating the isothermal microgaseous flows. The 2122method is also applied to the gas flow in a microchannel with sudden expansion/ contraction. Agrawal et al.²⁹ investigated the rarefied gas flow in a microchannel 23with sudden expansion or contraction in the slip regime using the LBM. Liou and 24Lin³⁰ also applied the LBM for simulating the microchannel gas flow with sudden 25expansion/contraction for a wide range of the Knudsen number. Recently, Li $et al.^{31}$ 26investigated the rarefied gas flow through channels with sudden and gradual con-2728traction junction with application in the shale gas flow using the multiple-relaxation-29time (MRT) LBM.

30 In order to simulate rarefied flows involving the off-lattice curved/inclined 31boundaries in LBM, Guo et al.³² approximated the curved boundary with the stairstep mesh. The approximated wall was taken to be located at half-way of each cut 3233 link, i.e. $\Delta = 0.5$. Furthermore, they implemented the Diffuse-Bounce-Back (DBB) 34 slip boundary condition and could simulate the rarefied cylindrical Couette flow and capture the velocity inversion phenomenon using the MRT collision operator. Later, 35Tao et al.³³ extended the DBB slip boundary condition for arbitrary values of 36 Δ using the MRT LBM. Recently, the sputtering boundary condition^{34,35} is devel-37 oped and utilized for determining amounts of reactants and products in a chemical 3839reaction occurring at solid boundaries of a catalyst.

In this paper, the TRT LBM proposed by Ginzburg³⁶ is utilized. It was reported
in the literature that values of the other relaxation times in the MRT LBM almost
have no influence on simulation results of the rarefied gas flows.³³ Therefore, it can be
concluded that the TRT LBM is robust enough for modeling such flows. Clearly, the

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1 TRT LBM takes advantage of a lower computational cost, rapid simulation time and 2 an easier coding. The TRT LBM was successfully applied by Norouzi *et al.*³⁷ for 3 simulating rarefied Poiseuille flow between two parallel plates in the entire range of 4 the transition regime.

5An inspection of the published literature reveals that application of the LBM for 6 simulating the rarefied gas flow through long diverging microchannels is not repor-7 ted. Here, the TRT LBM is employed to simulate divergent micro and nanochannels 8 for the first time. The divergent wall is approximated with the stair-step mesh, and 9 the Bounce-Back Specular Reflection (BSR) slip boundary condition is applied. The 10 power-law (PL) wall function is implemented to capture the Knudsen layer effects in 11 the transition regime. Pressure and Knudsen number distributions, slip, and cen-12terline Mach numbers, and mass flow rate are studied and compared with DSMC 13 results, where available, at different divergence angles. Furthermore, it is shown that 14there is a threshold divergence angle at a given pressure ratio where the unbounded 15Knudsen number and Mach number remain constant along the channel.

2. Numerical Model

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The governing equation of the TRT LBM is expressed as

$$f_i(\boldsymbol{x} + \boldsymbol{e}_i \delta t, t + \delta t) - f_i(\boldsymbol{x}, t) = -\frac{1}{\tau_s} (f_i^s(\boldsymbol{x}, t) - f_i^{\text{seq}}(\boldsymbol{x}, t)) - \frac{1}{\tau_a} (f_i^a(\boldsymbol{x}, t) - f_i^{\text{aeq}}(\boldsymbol{x}, t)),$$
(1)

where \boldsymbol{x} is the spatial position, \boldsymbol{e}_i the discrete velocity in the ith direction, δt the time step, f_i the particle distribution function, and sub/super-scripts s and a stand for symmetric and antisymmetric parts, respectively. Compared with the MRT relaxation times, $\tau_s = \tau_{\nu}$ and $\tau_a = \tau_q$, where τ_{ν} and τ_q are the relaxation times related to the shear stress and heat flux, respectively.

The equilibrium distribution function (EDF) is given by

$$f_i^{\rm eq} = \omega_i \rho \left[1 + \frac{\boldsymbol{e}_i \cdot \boldsymbol{V}}{c_s^2} + \frac{(\boldsymbol{e}_i \cdot \boldsymbol{V})^2}{2c_s^4} - \frac{\boldsymbol{V} \cdot \boldsymbol{V}}{2c_s^2} \right],\tag{2}$$

where $c_S = c/\sqrt{3}$ is the speed of sound, and $c = \delta x/\delta t$ is the lattice speed with δx as the lattice spacing.

36 The symmetric and antisymmetric distribution functions are calculated as 37 follows:

$$f_i^s = \frac{1}{2}(f_i + f_{-i}), \quad f_i^a = \frac{1}{2}(f_i - f_{-i}), \tag{3}$$

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$$f_i^{\text{seq}} = \frac{1}{2} (f_i^{\text{eq}} + f_{-i}^{\text{eq}}), \quad f_i^{\text{aeq}} = \frac{1}{2} (f_i^{\text{eq}} - f_{-i}^{\text{eq}}),$$
 (4)

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43 where -i is the opposite direction of i.

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For the D2Q9 lattice model used in this paper, the discrete velocities and the weight factors are given by

$$e_i = c \begin{bmatrix} 0 & 1 & 0 & -1 & 0 & 1 & -1 & -1 & 1 \\ 0 & 0 & 1 & 0 & -1 & 1 & 1 & -1 & -1 \end{bmatrix},$$
(5)

$$\omega_i = \begin{cases} 4/9, & i = 0, \\ 1/9, & i = 1, 2, 3, 4, \\ 1/36, & i = 5, 6, 7, 8. \end{cases}$$
(6)

The fluid macroscopic density, velocity and pressure are given by

$$\rho = \sum_{i} f_{i}, \quad \rho \mathbf{V} = \sum_{i} f_{i} \mathbf{e}_{i}, \quad P = \rho c_{s}^{2}.$$
(7)

The symmetric relaxation time is related to the Knudsen number as follows³⁸:

$$\tau_s = \sqrt{\frac{6}{\pi}} \frac{K n_e H}{\delta x} + 0.5, \tag{8}$$

where Kn_e is the effective Knudsen number and H local height of the channel in case of the divergence channel. Effects of the Knudsen layer in the transition regime is considered by implementing the PL wall function proposed by Dongari *et al.*,³⁹ which is reported to be applicable over a wide range of Knudsen number in the transition regime:

$$Kn_e = Kn\beta_{PL},\tag{9}$$

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$$\beta_{PL} = 1 - \frac{1}{96} \left[\left(1 + \frac{\frac{H}{2} - y}{\lambda} \right)^{-2} + \left(1 + \frac{\frac{H}{2} + y}{\lambda} \right)^{-2} \right]$$

$$+ 4\sum_{k=1}^{8} \left(1 + \frac{\frac{H}{2} - y}{\lambda \cos\frac{(2k-1)\pi}{32}}\right)^{-2} + 4\sum_{k=1}^{8} \left(1 + \frac{\frac{H}{2} + y}{\lambda \cos\frac{(2k-1)\pi}{32}}\right)^{-2} + 2\sum_{k=1}^{7} \left(1 + \frac{\frac{H}{2} - y}{\lambda \cos\frac{k\pi}{16}}\right)^{-2} + 2\sum_{k=1}^{7} \left(1 + \frac{\frac{H}{2} + y}{\lambda \cos\frac{k\pi}{16}}\right)^{-2}, \quad (10)$$

32 33where Kn is the unbounded Knudsen number. Effects of the channel walls (Kn layer) 34are not considered in the unbounded Knudsen number. Therefore, the Knudsen 35number needs to be corrected and the Kn layer effects included in the transitional 36 regime, which is accomplished by implementing the PL wall function. The effective 37 Knudsen number (Kn_e) is then calculated by modifying the unbounded Knudsen 38number given by Eq. (9). Indeed, the Kn_e governs the flow in the micro/nano-39channel. As mentioned, H is the height of the channel which is no longer constant in 40 case of the divergent channel, and y is the vertical coordinate of nodes measured from 41 the channel symmetry axis. The mean free path equals $\lambda = Kn(x) \cdot H(x)$. For the 42isothermal flow in the divergent channels, the following relation can be deduced for

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(12)

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Fig. 1. (Color online) A flat solid boundary with known (orange) and unknown (blue) distribution functions.

the local unbounded Knudsen number:

$$\lambda_i P_i = \lambda P \to K n_i H_i P_i = K n. H. P \to K n(x) = \frac{K n_i P_i}{P(x)} \frac{H_i}{H(x)}.$$
 (11)

In this paper, the BSR slip boundary condition is employed. For example, for the flat wall shown in Fig. 1, the half-way BSR is applied as follows:

 $f_7 = rf_5' + (1 - r)f_6',$

 $f_8 = rf_6' + (1 - r)f_5',$

 $f_4 = f'_2,$

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where f' denotes the post-collision distribution function and r determines the bounce-back portion. Clearly, setting r = 1 reduces the boundary condition to the half-way bounce-back and r = 0 to the specular wall.

24As schematically depicted in Fig. 2, the divergent wall is approximated with the 25half-way stair-step Cartesian mesh such that the approximated wall is located at the 26middle of each cut-link between the immediate fluid and solid nodes adjacent to 27the actual wall. Apparently, the approximated wall would better represent the actual 28one with a finer mesh resolution. 29

As observed in Fig. 2, the approximated wall consists of a collection of flat walls; 30 thus the BSR can still be applied easily with special cares of the nodes adjacent to the steps. The slip velocity as a result of applying the BSR boundary condition on a flat 32





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wall is given by³⁸:

$$U_s = \frac{4(1-r)}{r} \sqrt{\frac{6}{\pi}} Kn + \frac{2\varpi}{\pi(\tau_s - 0.5)^2} Kn^2,$$
(13)

where $\varpi = 16(\tau_s - 0.5)(\tau_a - 0.5) - 3$. On the other hand, the analytical slip velocity arising from applying the second-order slip boundary condition $u_s = L_1 \lambda \frac{\partial u}{\partial y}|_{\text{wall}} - L_2 \lambda^2 \frac{\partial^2 u}{\partial x^2}|_{\text{wall}}$ is obtained to be

$$U_s = 4L_1Kn + 8L_2Kn^2. (14)$$

10 Therefore, equating Eqs. (13) and (14) leads to the following relations:

$$r = \left(1 + \sqrt{\frac{\pi}{6}}L_1\right)^{-1},$$
(15)

$$\tau_a = \tau_q = 0.5 + \frac{4\pi L_2 (\tau_s - 0.5)^2 + 3}{16(\tau_s - 0.5)}.$$
(16)

Although these relations are extracted for the Poiseuille flow between two parallel plates, they can also be used for cases with curved or inclined surfaces.^{33,40}

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3. Results and Discussion

21A schematic drawing of the divergent channel is depicted in Fig. 3. The divergence angle is defined as $\beta = \tan^{-1} \frac{H_o - H_i}{L}$. The inlet height and the length of the channel are 2223fixed at $H_i = 12.5 \,\mathrm{nm}$ and $L = 250 \,\mathrm{nm}$ in lattice units, respectively, with a lattice 24spacing and time step of $\delta x = \delta t = 0.5$. The channel aspect ratio equals $L/H_i = 20$ 25which is taken as the same value as the DSMC simulations carried out by Ebrahimi 26and Roohi.¹² The inlet and outlet pressures are specified using the Zou and He⁴¹ 27pressure boundary condition. The symmetric boundary condition is applied on the 28symmetry axis. 29

Simulation results of the pressure distribution along the channel at an inlet/outlet pressure ratio of $\Pi = 2.5$, and inlet Knudsen number of $Kn_i = 1$ are validated with those of the DSMC in Fig. 4, where an excellent agreement is observed over a wide range of the divergence angle. Furthermore, the pressure distribution is observed to be almost independent of the slip coefficients. As observed, at a fixed pressure ratio, at





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Fig. 4. (Color online) Comparison of pressure distribution with DSMC results of Ref. 12, $\Pi = 2.5$, $Kn_i = 1$.

the pressure distribution becomes more concave as the divergence angle is increased.
This is because the pressure-driven flow is mainly opposed by the viscous forces and
the gradual increase of the channel height. Therefore, the pressure falls more rapidly
as the angle is increased.

Figure 5 shows the Mach number contour and velocity streamlines at three divergence angles with $\Pi = 2.5$, $Kn_i = 0.3$. As observed, while Knudsen number and as a result Mach number always increase along a straight microchannel in the isothermal pressure driven condition; see Fig. 5(a), an alike trend does not occur necessarily in case of the divergent microchannel where the behavior depends on the pressure ratio and divergence angle. According to values of these two parameters, three scenarios are found to occur in divergent microchannels that are; (1) Mach/Kn

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Fig. 5. (Color online) Mach number contours and streamlines at $\Pi = 2.5$ and $Kn_i = 0.3$.

increase similar to straight channels, (2) constant Ma/Kn condition, as shown in 30 Fig. 5(b), and (3) Ma/Kn decrease along the channel, see Fig. 5(c). Therefore, at 31each pressure ratio, there is a threshold divergence angle where increasing the di-32 vergence angles would lead to Ma/Kn reduction and decreasing the divergence angle 33 results in Ma/Kn increases. This angle is discussed and determined in the following 34 section of the paper. Furthermore, no flow separation occurs at $\beta = 16.7^{\circ}$ due to the 35slip velocity on the wall. 36

Variations of the slip and central Mach numbers distribution along the channel 37 for three divergence angles are shown in Figs. 6 and 7, respectively. As observed in 38 39Fig. 6, except for $\beta = 16.7$, the LBM results for the slip Mach number are in a good agreement with those of the DSMC only if L_2 is set relatively much smaller than L_1 . 40 In other words, the slip velocity should be dominantly adjusted by the value of r, as 41 was expected. Furthermore, the value of r has to be decreased with the increase in 42 the channel angle to produce higher slip velocities predicted by the DSMC results. 43

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Fig. 6. (Color online) Slip Mach number variations along the channel at $\Pi = 2.5$, $Kn_i = 1$.

31However, results of the slip velocity at $\beta = 16.7$ deviate considerably from the DSMC 32 solutions, even though the value of r is severely reduced. In other words, it was found 33 that at a fixed L₂ magnitude, the slip velocity at this relatively large angle approa-34 ches an asymptotic value with decreasing the value of r and the slip is no more 35increased considerably with further decrease of r. Therefore, it seems that the value 36 of L₂ should be increased to approach the DSMC results, but this will lead to 37 overprediction of the centerline Mach number. Therefore, it can be concluded that 38 the model fails to predict the slip velocity accurately at relatively high divergence angles. However, it will be shown and discussed in what follows that the mass 3940 flow rate is predicted very well by the model even at large divergence angles like 16.7 using the same r and L_2 values. From a practical point of view, accurate prediction 41 42 of the mass flow rate is probably the most crucial feature of a prevailing 43numerical solver.

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Fig. 7. (Color online) Centerline Mach number variations along the channel at $\Pi = 2.5$ and $Kn_i = 1$.

Distribution of the Knudsen number along the channel is shown in Fig. 8 for three 31angles. As observed, variations of the Knudsen number are consistent with those of 32 the slip and centerline Mach numbers at the corresponding angle. It is concluded that 33 the slip and centerline Mach numbers at $\beta = 4.3^{\circ}$ are nearly constant along the 34 channel due to a constant value of Kn along the channel. According to Eq. (11), 35Knudsen number remains constant because the pressure drop is compensated by the 36 increase in the channel height. The divergence angle at which the Knudsen number 37 remains constant is found to be dependent only on the pressure ratio. 38

Figure 9 shows variations of the divergence angle where Knudsen number is 39constant along the channel, β_c , with the pressure ratio. As observed, the angle 40 increases almost linearly by the pressure ratio. The linear curve fit is 41

$$\beta_c = 2.82\Pi - 2.7; \ 1.5 \le \Pi \le 6.$$

$$(17)$$

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Figure 10 shows variations of the non-dimensional mass flow rate, $M_n = \frac{L\sqrt{2RT}}{(P_i - P_o)H_i^2 w} \dot{M}$, with the mean Knudsen number, Kn_m , the latter is calculated at the average pressure of the inlet and outlet. As observed, a very suitable agreement is obtained until $Kn_m \approx 1$ with slip dominated coefficients, but they cannot predict the Knudsen minimum phenomenon. However, the Knudsen minimum is well captured by the model choosing a moderate L_2 value with a good agreement in the entire range of the transition regime. At $\beta = 16.7^{\circ}$, the Knudsen minimum is weak and the flow rate almost remains constant. Furthermore, as mentioned, even though the model fails to predict the slip velocity properly at this high opening angle, mass flow rate is







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 $\beta(^{\circ})$ L_2 Kn_m range Error_{max}% r6%0 0.580.2Entire transition 3%Entire transition 1.430.580.30.356 0.03 10%4.30.1 - 14.30.580.5Entire transition 13%70.20.03 0.1 - 18% 7 0.580.50.1 - 513%0.1 - 212%

29well estimated by the model. This is because the density (pressure) is considered in 30 the mass flow rate and the overprediction of the centerline Mach number is com-31pensated by the consequent density reduction due to the accurate prediction of the 32 pressure (density) distribution by the model independent of the divergence angle, as 33depicted in Fig. 4.

0.14

34 Based on the simulation results that are validated with DSMC solutions at 35 $\Pi = 2.5$, the slip coefficients given in Table 1 are suggested to predict the mass flow 36 rates with a suitable accuracy using the current LBM in a wide range of the Knudsen 37 number in the transition regime.

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4. Concluding Remarks

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The rarefied gaseous flow through diverging micro and nanochannels at different 41 42 operating conditions and divergence angles was investigated using the TRT LBM. 43 The BSR slip boundary condition was implemented coupled with the second-order S. F. Kharmiani & E. Roohi

1 slip boundary condition. The divergence wall was approximated with the so-called $\mathbf{2}$ stair-step Cartesian one and the power law wall function was implemented to capture 3 the Knudsen layer effects. Results were compared with those of DSMC reported in 4 the literature. According to the results, the following conclusions can be made for 5rarefied flows in diverging micro and nanochannels: 6 • The local pressure distribution has an excellent agreement with DSMC results in a 7 wide range of the divergence angle almost independent of the slip coefficients. 8 • There is a specific divergence angle at which the unbounded local Knudsen number 9 and therefore Mach number remain constant along the channel; the angle depends 10 only on the pressure ratio with an almost linear trend. 11 • The mass flow rate can well be predicted by the current TRT LBM model in a wide 12range of Kn and divergence angles with an appropriate choice of slip coefficients. 13 • The Knudsen minimum phenomenon is captured in the divergent channel only by 14 choosing proper slip coefficients. 151617Acknowledgment 18 The authors would like to acknowledge the financial support provided by the Faculty 19of Engineering, Ferdowsi University of Mashhad under Grant No. 47224. 202122References 231. Y. Zohar, S. Y. K. Lee, W. Y. Lee, L. Jiang and P. Tong, J. Fluid Mech. 472, 125 (2002). 242. C.-I. Weng, W.-L. Li and C.-C. Hwang, Nanotechnol. 10, 373 (1999). 253. N. Dongari, A. Agrawal and A. Agrawal, Int. J. Heat Mass Transf. 50, 3411 (2007). 264. A. Agrawal and A. Agrawal, Phys. Fluids, 18, 103604 (2006). 275. T. Ewart, P. Perrier, I. A. Graur and J. G. MéOlans, J. Fluid Mech. 584, 337 (2007). 6. E. Roohi and M. Darbandi, Phys. Fluids 21, 82001 (2009). 287. N. Singh, N. Dongari and A. Agrawal, Microfluid. Nanofluid. 16, 403 (2014). 298. F. Sharipov and G. Bertoldo, J. Vac. Sci. Technol. A Vacuum, Surfaces, Film. 23, 531 30 (2005).319. N. D. Stevanovic, J. Micromech. Microeng. 17, 1695 (2007). 3210. V. Hemadri, V. V. Varade, A. Agrawal and U. V. Bhandarkar, Phys. Fluids 28, 22007 33(2016).11. M. Akbari, A. Tamayol and M. Bahrami, J. Fluids Eng. 135, 71205 (2013). 34 12. A. Ebrahimi and E. Roohi, Microfluid. Nanofluidics, 21, 18 (2017). 3513. S. P. Kiselev, V. P. Kiselev and V. N. Zaikovskii, Shock Waves 28, 829 (2018). 36 14. M. Darbandi and E. Roohi, Int. Commun. Heat Mass Transf. 38, 1443 (2011). 37 15. A. M. Mahdavi, N. T. P. Le, E. Roohi and C. White, Numer. Heat Transf. Part A Appl. 38 **66**, 733 (2014). 16. A. M. Mahdavi and E. Roohi, Phys. Fluids 27 72002 (2015). 39 17. A. Gavasane, A. Agrawal and U. Bhandarkar, Vacuum, 155 249 (2018). 40 18. S. Chen and G. D. Doolen, Annu. Rev. Fluid Mech. 30, 329 (1998). 41 19. X. He and L.-S. Luo, Phys. Rev. E 56, 6811 (1997). 4220. P. Lallemand and L.-S. Luo, Phys. Rev. E 61, 6546 (2000). 43

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